

2101

Q1 - 1(a) - critical depth

a) For rectangular channel

$$y_c = \left(\frac{Q^2}{g} \right)^{1/3} = \frac{2}{3} \times E_c$$

$$y_c = \frac{2}{3} \times 1.5 = \underline{1 \text{ m}}$$

$$\boxed{y_c = 1 \text{ m}}$$

b) Triangular channel

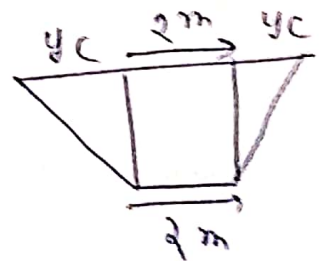
$$y_c = 0.8 \times E_c$$

$$\boxed{y_c = 1.2 \text{ m}}$$

c) Trapezoidal channel

$$E_c = y_c + \frac{V^2}{2g}$$

$$1.5 = y_c + \frac{Q^2}{A^2 \times 2g}$$



we know $F_r^2 = \frac{Q^2 T}{g A^3} = 1$

$$1.5 = y_c + \frac{1 \times A \times g}{T \times 2 \times g}$$

$$1.5 = y_c + \frac{0.5 (2 + 2 + 2y_c) \times y_c}{(2 + 2y_c) \times 2}$$

$$y_c = 1.095 \text{ m}$$

1) For a rectangular →

$$Q = \frac{2}{3} C_d L \sqrt{2g} H^{3/2}$$

$$1) Q = \frac{2}{3} \times 0.6 \times 50 \times \sqrt{2 \times 9.81} \times (0.5)^{3/2}$$

$$Q = 31.32 \text{ m}^3/\text{sec}$$

For Broad crested weir

$$1) Q = \frac{1.7 C_d L H^{3/2}}$$

$$Q = 1.7 \times 0.6 \times 50 \times (0.5)^{3/2}$$

$$Q = 18.03 \text{ m}^3/\text{sec}$$

$$1) \text{ velocity of approach } \Rightarrow V_a = \frac{18.03}{50} = \underline{\underline{0.361 \text{ m/sec}}}$$

$$h_a = \frac{V_a^2}{2g} \Rightarrow \underline{\underline{0.00663 \text{ m}}}$$

$$Q = 1.7 C_d L \left((H + h_a)^{3/2} - h_a^{3/2} \right)$$

$$Q = 18.363 \text{ m}^3/\text{sec}$$

Ans- 1-(c) → module requirements

- 1) It should have sufficiency capacity to carry discharge of distributory and water course discharge
- 2) control in flow depth
- 3) control in the diversion or passing of water
- 4) should have a higher stability
- 5) should have a sufficient durability.

Types of modules-

1) Non modular outlet -

→ discharge is depends on both the head of distributory and water course.

→ ex → submerged outlet, weir type outlet

2) semi-modular / flexible outlet

→ discharge depends upon ~~the~~ head of distributory channel ex → sub. pipe outlet, Kennedy's outlet

3) Rigid outlet -

→ discharge does not depends upon the head
difference b/w distributory and water course

→ ex → Khanna's outlet

2) Gibs' outlet

Q.1 (d) - Different types of pipes

1) vitified clay pipe -

→ These pipes are ~~low~~ acid resistance

→ mainly used when pipe carry huge amount of industrial waste in which there is high amount of acid present.

2) concrete pipes -

→ less corrosion resistance

→ difficulty in placing of concrete pipes

→ diameter is less

→ concrete pipes are affected by the acid flowing in sewerage

3) cast iron pipes →

→ light in weight

→ corrosion resistance

→ used for large diameter pipes

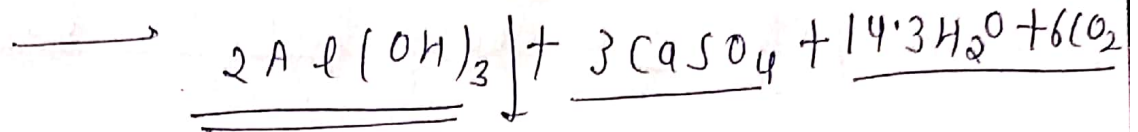
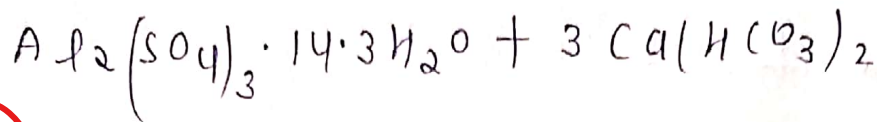
4) steel pipes -

- strength is good
- lesser corrosion resistance
- ease in placing of such pipes
- large diameter of such pipes can be used

5) Prestressed concrete pipes -

- Large strength
- difficulty in construction
- difficulty in laying down of such pipe

dry-solids -



1) alkalinity present

$$= \frac{3 \times 100}{599.4} \times \frac{50 \text{ mg}}{\text{l}}$$

$$= \underline{25.025 \text{ mg/l as } CaCO_3}$$

11) sludge produced -

$$\text{dry solids} \Rightarrow \frac{2 \times 156}{599.4} \times \frac{50 \text{ mg}}{\text{l}} \times 10^{-6} \times 5 \times 10^7 \times 0.65$$

$$= \underline{1301.30 \text{ kg/day}} \times 0.65 = \underline{845.85 \text{ kg/day}}$$

$$\text{volume of sludge produce} = \frac{1301.30 \times 0.65}{1040}$$

$$= \underline{\underline{0.8133 \text{ m}^3/\text{day}}}$$

Ans- 3(a) - shaft power = 7357.5 KW

overall efficiency $\eta_o = \frac{S.P}{W.P}$

$W.P = \frac{7357.5}{0.6}$

$\gamma Q H = \frac{7357.5}{0.6}$

$9.81 \times Q \times 5.5 = \frac{7357.5}{0.6}$

$Q = 227.27 \text{ m}^3/\text{sec}$

speed ratio = 2.09

speed of wheel = $2.09 \sqrt{2 \times 9.81 \times 5.5}$

$U = 21.71 \text{ m/sec}$

velocity of flow $V_f = 0.68 \sqrt{2 \times 9.81 \times 5.5}$

$V_f = 7.064 \text{ m/sec}$

discharge $Q = \frac{\pi}{4} (D_R^2 - D_b^2) \times 7.064$

$227.273 = \frac{\pi}{4} \times \left(D_R^2 - \frac{1}{9} D_R^2 \right) \times 7.064$

$D_R = 6.78 \text{ m}$

$$\text{dia of runner } D_R = \underline{6.78 \text{ m}}$$

$$\text{dia of boss} = \frac{1}{3} \times 6.78 = \underline{2.26 \text{ m}}$$

we know $U = \frac{\pi D_R N}{60}$

$$21.71 = \frac{\pi \times 6.78 \times N}{60}$$

$$N = 61.158 \text{ rpm}$$

specific speed $N_s = \frac{N \sqrt{Q}}{H^{3/4}}$

$$N_s = \frac{61.15 \sqrt{227.27}}{(5.5)^{3/4}}$$

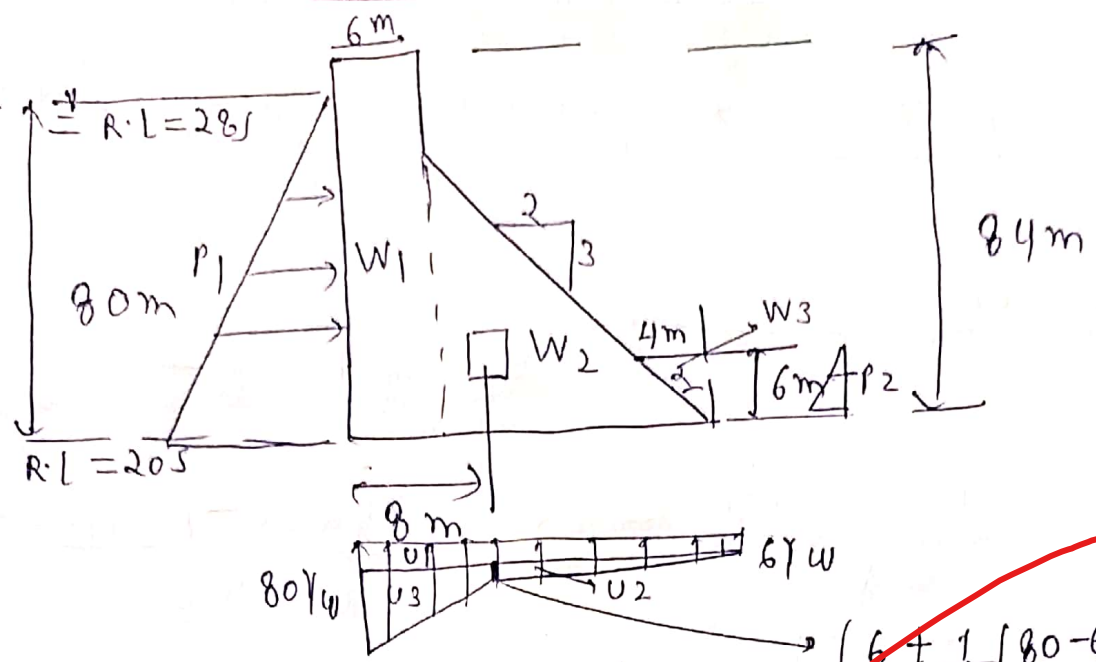
$$N_s = 256.708 \text{ rpm}$$

In term of power $N_s = \frac{N \sqrt{P}}{H^{5/4}}$

$$N_s = \frac{61.15 \sqrt{7357.5}}{(5.5)^{5/4}}$$

$$N_s = 622.748 \text{ rpm}$$

3 (b)



Force (KN)

$W_1 \downarrow$

calculation $6 \times 84 \times 23.5 = \underline{11844 \text{ KN}}$

$(6 + \frac{1}{3}(80-6)) \gamma_w$
 $= \frac{92}{3} \gamma_w \frac{(L \cdot A)}{(m)}$
 (about toe)
 53 m

$W_2 \downarrow$

$0.5 \times 50 \times 75 \times 23.5 = 44062.5 \text{ KN}$

100/3

P_1
 (\rightarrow)

$\frac{1}{2} \times 9.81 \times (80)^2 = 31392 \text{ KN}$

80/3

P_2

$\frac{1}{2} \times 9.81 \times (6)^2 = 176.58 \text{ KN}$

6/3 = 2

$U_1 \uparrow$

$6 \times 9.81 \times 56 = 3296.16 \text{ KN}$

28

$U_2 \uparrow$

$24.67 \times 9.81 \times 0.5 \times 48 = 5808.30 \text{ KN}$

32

$U_3 \uparrow$

$0.5 (74 + 24.67) \times 9.81 \times 8 = \underline{3871.68 \text{ KN}}$

52/3

$W_3 \downarrow$

$0.5 \times 4 \times 6 \times 9.81 = \underline{117.72 \text{ KN}}$

4/3

$$\Sigma V = \underline{42930.36 \text{ kN}} + 117.72 = \underline{43048.08 \text{ kN}}$$

$$\rightarrow \text{max}^m \text{ vertical stress} = \frac{\Sigma V}{B} \left(1 + \frac{6e}{B} \right)$$

$$\bar{x} = \frac{\Sigma M_R - \Sigma M_o}{\Sigma V} = \frac{2096992.12 - 1319160.75}{43048.08}$$

$$\boxed{\bar{x} = 18.07 \text{ m}}$$

$$e = \frac{B}{2} - \bar{x} = \underline{9.93 \text{ m}}$$

max^m vertical stress at toe

$$\sigma = \frac{43048.08}{56} \left(1 + \frac{6 \times 9.93}{56} \right)$$

$$\boxed{\sigma = 1586.57 \text{ kN/m}^2} < 2500 \text{ kN/m}^2$$

max^m vertical stress at heel

$$\sigma = \frac{43048.08}{56} \left(1 - \frac{6e}{B} \right)$$

$$\boxed{\sigma = -49.24 \text{ kN/m}^2} \rightarrow \text{tension}$$

ii) major principle stress

$$= \frac{\sigma_v \sec^2 \alpha}{\phi'} - \tan^2 \alpha$$

$$\tan \alpha = 2/3$$

$$\sec \alpha = 1.2018$$

$$\text{major principle stress} = 1586.57 \times (1.2018)^2 - 9.81 \times 6 \times (2/3)^2$$

$$\sigma_{\text{max}} = 2265.36 \text{ KN/m}^2$$

iii) Intensity of shear stress

$$\tau = (\sigma_v - \sigma_l) \tan \alpha$$

$$\tau = (1586.57 - 9.81 \times 6) \times 2/3$$

$$\tau = 1018.47 \text{ KN/m}^2$$

20

Q3-3-(c) -

Population = 1,00,000

~~Total discharge = $180 \times 10^5 = \underline{\underline{18 \text{ MLD}}}$~~

~~$\frac{q_{\text{design}}}{q_{\text{max}}} = \frac{1}{3} = \underline{\underline{0.3333}}$~~

~~Proportionate velocity for proportion discharge~~

~~of $\frac{1}{3}$~~

~~$\frac{v}{V} = 0.8909 + \frac{0.9022 - 0.8909}{0.3370 - 0.3217} (0.3333 - 0.3217)$~~

~~$\frac{v}{V} = 0.899$~~

Proportionate depth

~~$\frac{d}{D} = 0.39 + \frac{0.40 - 0.39}{0.3370 - 0.3217} (0.3333 - 0.3217)$~~

~~$\frac{d}{D} = 0.3976$~~

$$3-(c)(ii) \text{ Total dry sludge} = 0.068 \times 40,000 \\ = \underline{2720 \text{ Kg}}$$

total Incoming sludge volume

$$= \frac{100 \times 2720}{6 \times 1020} = \underline{44.44 \text{ m}^3}$$

$$\frac{3.5}{100} \times V_{\text{digester}} = 44.44$$

volume of digester $V = 1269.84 \text{ m}^3$

Take $H = \underline{3 \text{ m}}$

$$\frac{\pi}{4} \times D^2 \times 3 = 1269.84$$

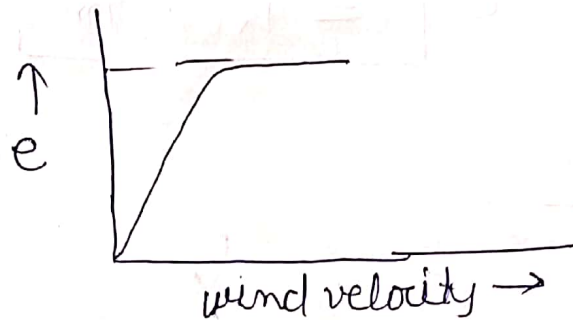
$$D = 23.215 \text{ m}$$

Provide $D = 24 \text{ m}$
 $H = 3 \text{ m}$ } rdigester disession

Q4 - 4(a) - Factor affecting evaporation
(i)

1) Temp → as the temp increases evaporation
also increases

2) wind velocity - as the wind velocity increases
the evaporation also increases upto certain
limit and then it becomes const



3) salinity - if water is saline, then evaporation
from it will be lesser than the normal water

4) surface area -
if more surface area is exposed to atmosphere
more evaporation will be there.

5) humidity - if humidity is high in atm.
than the evaporation will be
lesser

6) Topography and hydrography of the area

7) depth of water in stream

In summer season there is an higher evaporation from the shallow stream

In winter season there is higher evaporation from the deeper stream

8) Pressure - Increase in pressure increases the evaporation upto certain limit

4-(a) (ii)

Evaporation Pan method

In this method there is a pan, in which there is water is filled, and it is placed in free atmosphere and evaporation is determined.

→ with the help of this pan we can calculate the evaporation from lake

$$\text{lake evaporation} = C_p \times \text{Pan evaporation}$$

$C_p \rightarrow$ Pan coefficient

$C_p \rightarrow 0.7 \rightarrow A, L \text{ type}$
 $\rightarrow 0.8 \rightarrow \text{IST}$

b) analytical method

i) water budget method -

Total inflow - Total outflow = Change in storage

$$\left(\frac{I_1 + I_2}{2} \right) \times \Delta t - \left(\frac{O_1 + O_2}{2} \right) \times \Delta t = S_2 - S_1$$

ii) Energy Balance method

iii) mass transfer approach

$$\text{Ay-4(b) - water supply} = 130 \times 50,000 \\ = \underline{6500 \text{ m}^3/\text{day}}$$

$$\text{Total sewage} = 0.8 \times 6500 = \underline{5200 \text{ m}^3/\text{day}}$$

$$\text{man}^m \text{ design sewage} = \frac{3 \times 5200}{86400} = \underline{0.1806 \text{ m}^3/\text{sec}}$$

$$\boxed{Q = 0.1806 \text{ m}^3/\text{sec}}$$

$$\text{velocity of flow in rising main} = 1.2 \text{ m/sec}$$

$$\text{Area of pipe main} = \frac{0.1806}{1.2} = \underline{0.1505 \text{ m}^2}$$

$$\frac{\pi}{4} \times D^2 = 0.1505$$

$$\boxed{D = 0.437 \text{ m}} \quad \text{or} \quad \underline{440 \text{ mm}}$$

$$\text{P inside dia of rising main} = \underline{440 \text{ mm}}$$

$$\text{ii) B.H.P required} = \frac{\gamma Q H_{\text{total}}}{\eta \times 0.746}$$

$$H_{\text{total}} = (205 - 195) + 0.3 + h_f \rightarrow \left(\frac{f L V^2}{2gD} \right) \\ = 10 + 0.3 + \frac{0.01 \times 100 \times (1.2)^2}{2 \times 9.81 \times 0.437} = \underline{10.47 \text{ m}}$$

$$\text{B.H.P required} = \frac{9.81 \times 0.1806 \times 10^4 \times 7}{0.746 \times 0.6}$$

$$\text{BHP required} = \frac{41.434 \text{ H.P}}{1}$$

Ans- 4-(c) - $B = 3.5 \text{ m}$ / $y = 2 \text{ m}$

$Q = 15 \text{ m}^3/\text{sec} \rightarrow F_r < 1 \rightarrow \text{subcritical}$

specific energy at u/s $E_1 = y_1 + \frac{v_1^2}{2g}$

$$E_1 = 2 + \frac{\left(\frac{15}{3.5 \times 2}\right)^2}{2 \times 9.81} = \underline{2.234 \text{ m}}$$

1) width contraction - $B_2 = 2.5 \text{ m}$

$$y_c = \left(\frac{Q^2}{g}\right)^{1/3} = \left(\frac{(15)^2}{(2.5) \times 9.81}\right)^{1/3} = 1.542 \text{ m}$$

$$E_c = \frac{3}{2} \times y_c = \underline{2.314 \text{ m}}$$

as $E_1 < E_c \rightarrow$ so can't possible

so we have to increase the depth at u/s

secⁿ

$$E_1' = E_c$$

$$y_1' + \frac{\left(\frac{1.5}{3.5 \times y_1}\right)^2}{2 \times 9.81} = 2.314$$

$$y_1' = 2.314 - \frac{0.93615}{y_1'^2}$$

$$\boxed{y_1' = 2.102 \text{ m}} \quad \cancel{0.208 \text{ m}}$$

so u/s depth of water = 2.102 m

d/s depth " = 1.542 m

ii) when $B_2 = 2.2 \text{ m}$

$$E_c = \left(\frac{v^2}{g}\right)^{1/3} \times \frac{3}{2} = \left(\frac{\left(\frac{1.5}{2.2}\right)^2}{9.81}\right)^{1/3} \times 1.5$$

$$\boxed{E_c = 2.52 \text{ m}} > \underline{E_{at \ u/s}}$$

so u/s depth has to increase

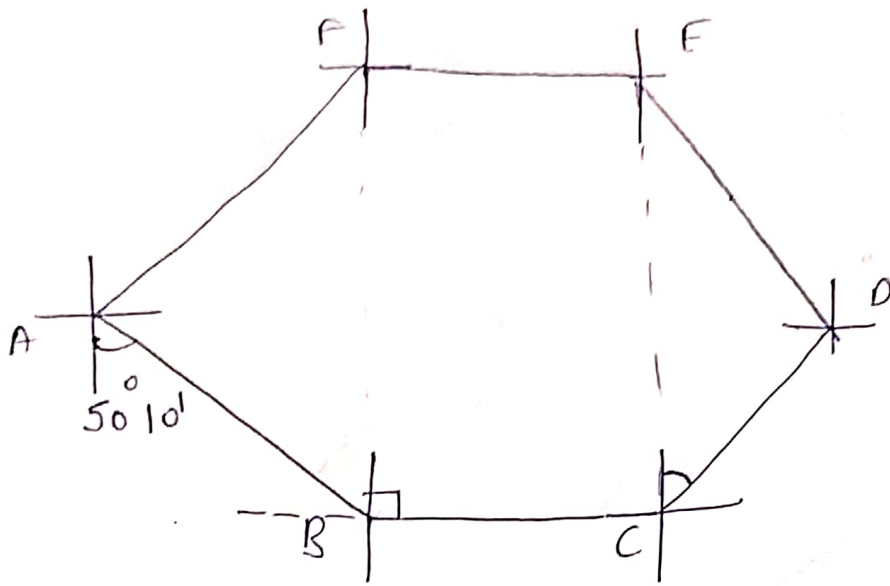
$$E_1' = E_c$$

$$y_1' + \frac{0.93615}{y_1'^2} = 2.52$$

$$\boxed{y_1' = 2.35 \text{ m}} \rightarrow \text{u/s water depth}$$

$$\underline{y_c = 1.68 \text{ m}} \rightarrow \text{d/s water depth}$$

Ques- 5(a)



Bearing of side AB = $180^\circ - 50^\circ 10'$ = $129^\circ 50'$

Bearing of side BC = 90°

Bearing of side CD = $50^\circ 10'$

Bearing of DE = $360^\circ - 50^\circ 10'$ = $309^\circ 50'$

Bearing of EF = 270°

Bearing of FA = $230^\circ 10'$

Q. 5 (b) -

fos against sliding

$$FOS = \frac{mR}{m_0}$$

$$FOS = \frac{C_u \times R^2 \times \theta}{W \times e}$$

$$FOS = \frac{20 \times (9)^2 \times \left(71^\circ \times \frac{\pi}{180^\circ} \right)}{329 \times 4.8}$$

$$FOS = 1.27$$

ii) If sliding portion removed

Qy-5-(c)- Length, $R = 220\text{m}$, $V = 65\text{kmph}$
 $= 18.06\text{m/s}$

Based on rate of change of acceleration

$$L = \frac{v^3}{cR} \Rightarrow$$

$$c = \frac{80}{75+v} = \frac{80}{75+65} = 0.5714$$

$$L = 46.82\text{m}$$

ii) Based on introduction of superlevation

$$L = \frac{N e W}{2}$$

design of superlevation

$$e = \frac{v^2}{225R} = \frac{(65)^2}{225 \times 220}$$

$$e = 0.085 \neq \underline{0.07}$$

so take $e = 0.07$

$$e + u = \frac{v^2}{gR}$$

$$u = 0.081 < 0.15 \text{ ok}$$

so take $e = 0.07 \rightarrow$ Plain and rolling condition

$$L = \frac{150 \times 0.07 \times 7.5}{2} = \underline{39.375 \text{ m}}$$

iii) length based on empirical

$$L = \frac{2.7 V^2}{R} = \frac{2.7 \times (65)^2}{220} = \underline{51.85 \text{ m}}$$

so provide length of transition curve = 51.85 m

$$\boxed{L = 52 \text{ m}}$$

$$\text{shift of curve } s = \frac{L^2}{24 R} = \frac{(52)^2}{24 \times 220} = \underline{0.51 \text{ m}}$$

Adv - 5(d) -

Based on martin's formula

$$V_{\text{max}} = 4.58 \sqrt{R}$$

$$V_{\text{max}} = 4.58 \sqrt{R}$$

$$\text{Radius } R = \frac{1720}{1} = \underline{1720 \text{ m}}$$

$$V = 4.58 \sqrt{1720} = \underline{189.95 \text{ km/hr}}$$

$$ii) e = \frac{W V_{avg}^2}{127 R}$$

$$\frac{80}{1000} = \frac{1.67 \times V^2}{127 \times 1720}$$

$$\boxed{V_{avg} = 102.29 \text{ Km/hr}}$$

$$iii) e_{act} = e_{equil} - c.d$$

$$e_{mon} = 80 + 100$$

$$\underline{e_{mon} = 180 \text{ mm}}$$

$$\frac{180}{1000} = \frac{1.67 \times V_{mon}^2}{127 \times R}$$

$$\boxed{V_{mon} = 153.44 \text{ Km/hr}}$$

$$\rightarrow \text{permissible mon}^m \text{ speed} = \underline{153.44 \text{ Km/hr}}$$

Based on Length of curve

$$V = \frac{198 L}{e} = \frac{198 \times 120}{80} = 297 \text{ Km/hr}$$

$$= \frac{198 L}{c.d} = \frac{198 \times 120}{100} = \underline{\underline{237.6 \text{ Km/hr}}}$$

$$V_{\text{mean}} = 153.44 \text{ km/hr} \rightarrow \text{mean permissibility speed}$$

Qdy = 5(c) - avg discharge for 2 min

$$= \frac{(541 + 503 + 509 + 474) \times 10^{-6}}{4 \times 60 \times 4}$$
$$= \underline{4.22 \times 10^{-6} \text{ m}^3/\text{sec}}$$

avg difference in head

$$= \frac{(76 + 72 + 68 + 65)}{4} = \underline{70.25 \text{ mm}}$$

$$Q = k i A$$

$$4.22 \times 10^{-6} \times 1 = k \times \frac{70.25}{150} \times \frac{\pi}{4} \times (0.1)^2$$

$$\boxed{k = 1.147 \times 10^{-3} \text{ m/sec}}$$

temp correction = 1.09

$$\underline{k = 1.25 \times 10^{-3} \text{ m/sec}}$$

Q.7 (a) - (i)

For at depth of 1 m

1) safe pressure - $\frac{c N_c + \gamma D_f}{FOS} + \gamma D_f$

$$\frac{1.1 \times 10^3}{B \times 1} = \frac{C N_c + (\gamma D_f) + 0.5 B \gamma N_c + \gamma D_f}{FOS}$$

$$\frac{1100}{B} = \frac{18 \times 22.3 + (11.9 - 1) \times 19 \times 1 + 0.5 \times B \times 19 \times 9.53}{3} + 19 \times 1$$

$$\frac{1100}{B} = \frac{1331}{3} + 30.178 B$$

$$B = 3.39 \text{ m}$$

ii) when $d_f = \underline{2 \text{ m}}$

$$\frac{1100}{B} = \frac{18 \times 22.3 + 10.9 \times 19 \times 2 + 0.5 \times B \times 19 \times 9.53}{3}$$

$$+ 19 \times 2$$

$$\frac{1100}{B} = 309.87 + 30.178 B$$

$$B = 2.79 \text{ m}$$

$$7 \rightarrow (a) (ii) \quad \underline{\text{embankment}} - \quad \gamma_d = 18 \text{ kN/m}^3$$

$$\underline{\text{Borrow pit}} - \quad \gamma_b = 17 \text{ kN/m}^3 \quad \left| \quad \gamma_d = \frac{\gamma_b}{1+w} \right)$$

$$(\gamma_d)_{br} = \frac{17}{1+0.05} = \underline{16.19 \text{ kN/m}^3}$$

void ratio from of embankment soil

$$e_1 \rightarrow \text{we know } (\gamma_d)_1 = \frac{w\gamma_w}{1+e_1}$$

$$18 = \frac{2.7 \times 9.81}{1+e_1}$$

$$\boxed{e_1 = 0.4715}$$

void ratio of borrow pit soil $\rightarrow e_2$

$$(\gamma_d)_2 = \frac{w\gamma_w}{1+e_2}$$

$$e_2 = \frac{2.7 \times 9.81}{16.19} - 1$$

$$\boxed{e_2 = 0.636}$$

$$\text{volume of solids } v_s = \frac{v}{1+e}$$

volumes of solids will be constant

$$\frac{v_1}{1+e_1} = \frac{v_2}{1+e_2}$$

$$\frac{1}{1+0.4715} = \frac{V_2}{1+0.636}$$

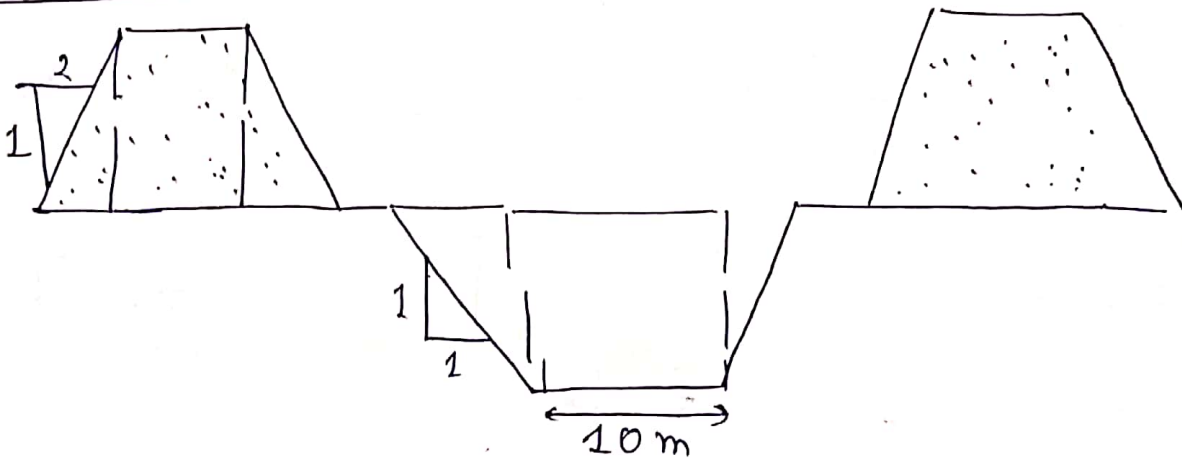
$$V_2 = 1.112 \text{ m}^3$$

swell factor = 1.2

volume required from borrow pit

$$= 1.2 \times 1.112 = \underline{1.334 \text{ m}^3}$$

Ans- 7(b) -



Qy - 7 - (c) - (ii)

$$1) \text{ Radius } R = \frac{V^2}{125f}$$

$$R = \frac{(60)^2}{125 \times 0.15} = \underline{192 \text{ m}}$$

$$ii) \quad R = \frac{0.388 W^2}{\frac{T}{2} - S}$$

$$S = 6 + \frac{7.5}{2} = \underline{9.75 \text{ m}}$$

$$T = 22.5 \text{ m}$$

$$R = \frac{0.388 \times (35)^2}{\frac{22.5 - 9.75}{2}} = \underline{316.86 \text{ m}}$$

Radius $R = \max^m (192, 180 \text{ m}, 316.86 \text{ m})$

So provide $R = \underline{317 \text{ m}}$

Q4-7-(c) (ii)

For A type -

$$\text{capacity} = \frac{5}{0.3 \times 60} \Rightarrow \underline{16.66 \text{ air/hr}}$$

For aircraft B -

$$\text{capacity} = \frac{3 \times 60}{0.5 \times 45} = \underline{8 \text{ aircraft/hr}}$$

For aircraft C type -

$$\text{capacity} = \frac{2 \times 60}{0.2 \times 30} = \underline{20 \text{ aircraft/hr}}$$

so aircraft capacity = ~~16.6~~

$$= \underline{8 \text{ aircraft/hr}}$$