

DETAILED EXPLANATIONS

- **1. (d)**
- **2. (c)**

4 points of contra-flexure.

FBD:

Applied moment at

kNm

4. (c)

$$
M_{FCB} = -\frac{12 \times 4^2}{12} = -16
$$

$$
M_{CB} + M_{CA} = 0
$$

$$
\Rightarrow -16 + \frac{2EI}{4} \times (2\theta_C) + \frac{2EI}{4} \times (2\theta_C) = 0
$$

$$
\Rightarrow \qquad \theta_C = \frac{8}{EI}
$$

5. (d)

For static equilibrium in a space structure equations to be satisfied are,

$$
\Sigma F_x = 0, \Sigma F_y = 0, \Sigma F_z = 0
$$

$$
\Sigma M_x = 0, \Sigma M_y = 0, \Sigma M_z = 0
$$

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- **6. (c)**
- **7. (d)**
- **8. (b)**
- **9. (d)**

Stiffness of beam,
$$
k_b = \frac{152}{l^3}
$$

$$
k_b = L^3
$$

$$
k_b = \frac{48 \times 1}{2^3} = 6
$$
unit

48*EI*

Equivalent stiffness, k_{eq}

$$
\frac{1}{k_{eq}} = \frac{1}{6} + \frac{1}{12} = \frac{3}{12}
$$

(Because, beam and spring are in series as there deflection is different)

 k_{eq} = 4 unit

10. (b)

The maximum bending moment for any position of load occur under the load. So the equation for maximum bending moment is

$$
M_{max} = \frac{x(L-x)}{L}
$$

Thus the ILD for maximum bending moment is parabolic. This is also called envelop of maximum bending moment.

11. (b)

Members of truss can be of different cross-section.

12. (c)

Let the vertical reaction at *A* and *B* be V_A and V_B respectively are horizontal thrust be *H*.

$$
\Sigma M_A = 0
$$
\n
$$
\Rightarrow \qquad V_B \times 12 - 8 \times P = 0
$$
\n
$$
\Rightarrow \qquad V_B = \frac{2P}{3}
$$
\n
$$
\Rightarrow \qquad V_A = P - \frac{2}{3}P = \frac{P}{3}
$$
\n
$$
\Sigma M_C = 0
$$
\n
$$
\Rightarrow \qquad 6 \times V_A - 4 \times H = 0
$$
\n
$$
\Rightarrow \qquad H = \frac{6V_A}{4}
$$
\n
$$
\Rightarrow \qquad H = \frac{6 \times P}{4 \times 3} = \frac{P}{2}
$$

13. (b)

At joint *B*,

Fixed end moments, M

$$
I_{FBC} = -\frac{W L}{8}
$$

$$
M_{FCB} = +\frac{WL}{8}
$$

Distributing ($-M_{FBC}$) to M_{BA} and M_{BC} . Final moment will be,

$$
M_{BA} = \frac{3}{7} \times \frac{WL}{8}
$$

$$
M_{BC} = -\frac{WL}{8} + \frac{4}{7} \times \frac{WL}{8} = -\frac{3WL}{56}
$$

and carryover of distributed moment of member *BC* is

So, final moment,
\n
$$
M_{CB} = \frac{1}{2} \left(\frac{4}{7} \times \frac{WL}{8} \right)
$$
\n
$$
M_{CB} = \frac{WL}{8} + \frac{2WL}{56} = \frac{9WL}{56}
$$
\n
$$
M_{BC} = \frac{-\frac{3WL}{56}}{-\frac{56}{56}} = -\frac{1}{1}
$$

M $\frac{16}{M_{CB}}$ =

So, $\frac{M_{BC}}{M_{CB}}$

Magnitude = 1 3

14. (b)

For two-hinged semi-circular arch with load *W* applied at any section, the radius vector corresponding to which makes an angle θ with the horizontal.

 $\frac{9WL}{ }$ 3

= −

56

WL

$$
H = \frac{W}{\pi} \sin^2 \theta
$$

With load at crown, $\theta = \frac{\pi}{2}$
So, $H = \frac{W}{\pi}$

15. (c)

H

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Circular frequency for damped condition is

$$
\omega_D = \omega_n \sqrt{1 - (\epsilon)^2}
$$

$$
= \sqrt{\frac{k}{m}} \times \sqrt{1 - \epsilon^2}
$$

$$
\omega_D = \sqrt{\frac{21 \times 10^3}{32}} \times \sqrt{1 - (\frac{3}{100})^2}
$$

$$
\omega_D = 25.6 \text{ rad/sec}
$$

$$
f_D = \frac{\omega_D}{2\pi} = \frac{25.6}{2\pi} = 4.075 \text{ Hz}
$$

16. (a)

Vertical reaction;
$$
V = \frac{wl}{2} = \frac{15 \times 150}{2} = 1125 \text{ kN}
$$

Horizontal reaction; $H = \frac{wl^2}{8h} = \frac{15 \times 150^2}{8 \times 10} = 4218.75 \text{ kN}$
 \therefore Maximum tension = $T_{\text{max}} = \sqrt{V^2 + H^2} = \sqrt{1125^2 + 4218.75^2} = 4366.17 \text{ kN}$
Minimum tension = $T_{\text{min}} = H = 4218.75 \text{ kN}$
 $T_{\text{max}} - T_{\text{min}} = 147.42 \text{ kN}$

$$
17. (a)
$$

For member *ACB,*

Moment at *X*,

\n
$$
M_x = PR - PR(1 - \cos \theta) = PR \cos \theta
$$
\nStrain energy stored,

\n
$$
W_i = \int \frac{M_x^2 ds}{2EI} = \int_0^\pi \frac{(PR \cos \theta)^2 Rd\theta}{2EI}
$$

$$
\Rightarrow \qquad W_i = \frac{P^2 R^3}{2EI} \times 2 \int_0^{\frac{\pi}{2}} \cos^2 \theta \, d\theta = \frac{P^2 R^3}{EI} \times \frac{\pi}{4}
$$

∴ Vertical deflection at *O* due to member *ACB*,

$$
\delta = \frac{\partial W_i}{\partial P} = \frac{\pi}{2} \frac{PR^3}{EI}
$$

For member OB,

Strain energy stored, *Wi* = $2d_c$ $\frac{R}{2}(D_v)^2$ 0 (Px) 2 EI $\frac{1}{0}$ 2 $M_x^2 ds - \int_{0}^{R} (Px)^2 dx$ $\int \frac{W_1 x^{1/3}}{2EI} = \int_0^{\infty} \frac{(1-x)^{1/3}}{2EI}$

$$
W_i = \frac{P^2 R^3}{6EI}
$$

∴ Vertical deflection at *O* due to member *OB*,

$$
\delta = \frac{\partial W_i}{\partial P} = \frac{1}{3} \frac{PR^3}{EI}
$$

Total deflection =
$$
\left(\frac{\pi}{2} + \frac{1}{3}\right) \frac{PR^3}{EI} = 1.904 \frac{PR^3}{EI}
$$

18. (d)

Apply a unit load at joint *E*

A
\nA
\n
$$
\delta_E = \Sigma K (L \alpha \Delta T)
$$
\nA t joint A;
\n
$$
\Sigma F_y = 0
$$
\n
$$
K_{AB} = \frac{1}{\sqrt{3}}
$$
 (compressive)
\nAlso;
\n
$$
K_{AB} = K_{CD}
$$
 (Due to symmetry)
\nFrom (1)
\n
$$
\delta_E = \left(\frac{1}{\sqrt{3}} \times 2 \times 12 \times 10^{-6} \times 20\right) \times 2 \text{ m} = 0.55 \text{ mm}
$$
\nA
\n
$$
\delta_E = \left(\frac{1}{\sqrt{3}} \times 2 \times 12 \times 10^{-6} \times 20\right) \times 2 \text{ m} = 0.55 \text{ mm}
$$

19. (d)

Cut a section through *BC*, *BE* and *EF*

$$
\Sigma F_y = 0; \implies 20 + 20 + F_{EF} \cos \theta + F_{BC} = 0 \qquad ...(1)
$$

\n
$$
\Sigma M_E = 0; \implies F_{BC} \times 8 + 20 \times 8 = 40 \times 8
$$

\n
$$
\implies F_{BC} = 20 \text{ kN}; \text{ (sin } \theta = \cos \theta = \frac{1}{\sqrt{2}}) \qquad (\because \theta = 45^\circ)
$$

From (1); $F_{EF} = -60\sqrt{2} \text{ kN}$ (–ve i.e. compression)

∴ Magnitude of F_{EF} = 60 $\sqrt{2}$ kN

20. (b)

Since, more moment will get transferred to the fixed ends due to decrease in stiffness of middle half.

21. (c)

As per Muller Breslau principle, release the reaction at A and a unit displacement is given to primary structure at A.

22. (b)

Since beam is symmetric

- ∴ Vertical reaction at *A* and *B* is i.e. $V_A = V_B = 50$ kN
- ∴ Taking moments about *C*

$$
M_A = V_A \times 4 = 50 \times 4 = 200 \text{ kNm}
$$

23. (c)

• Due to load

$$
M_{FBA} = \frac{VVL}{8}
$$
 (clockwise)

• Due to sinking of support

$$
= \frac{VVL}{8}
$$

So fixing moment at *B*,

$$
M_{FBA} = \frac{WL}{8}(ACW) + \frac{WL}{8}(CW)
$$

= 0

24. (b)

Due to sinking of support *A*,

Hence, reaction at *C* decreases.

25. (b)

26. (a)

ILD for SF at *C* is shown below

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$$
\therefore \qquad \text{Maximum S.F.} = \frac{1}{2} \times 0.6 \times 6 \times 15
$$

$$
= 27 \text{ kN}
$$

27. (b)

When head of UDL is at 8 m from *A*,

SF =
$$
40\left[\frac{4}{20} + \frac{8}{20}\right] \times \frac{1}{2} \times 4 = 48 \text{ kN}
$$

28. (b)

 $j = (S + 1) (B + 1) =$ Number of joints $m = S(B + 1) + BS =$ Number of members $r_e = 3(B + 1)$ $D_k = 3j - r_e - m$ ∴ $D_k = 3(S + 1) (B + 1) - 3(B + 1) - S(B + 1) - BS$ $= S(B + 2)$

29. (b)

30. (d)

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