

POSTAL **Book Package**

2023

Mechanical Engineering

Conventional Practice Sets

Engineering Mechanics

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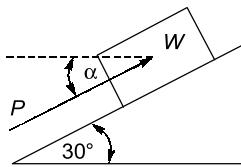
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Equilibrium of Forces and Moment

Practice Questions

- Q1** Determine the magnitude and direction of the smallest force P , which will maintain the body of weight $W = 300 \text{ N}$ on an inclined smooth plane as shown in figure is in equilibrium.



Solution:

The body is acted upon by three forces, namely the action of gravity force W , the applied force P and the reaction R . Since these three forces are in equilibrium, the vectors representing them must build a closed triangle, we begin with the known vector \overline{bc} representing to a certain scale, the weight of the body, and then draw the line aa' parallel to the R .

The side \overline{cd} will be minimum if it is perpendicular to line aa' ,

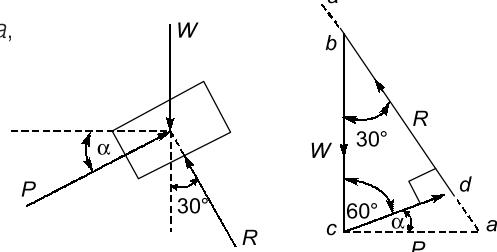
that is P will be minimum, if it is perpendicular to aa' .

From the triangle bcd , $\angle c = 90^\circ - 30^\circ = 60^\circ$

$$\therefore \alpha = 90^\circ - 60^\circ = 30^\circ$$

and using the triangle bcd , we obtain,

$$P = W \sin 30^\circ = \frac{W}{2} = 150 \text{ N}$$



Alternate solution: After drawing the free-body diagram of the body of above, then applying the Lami's theorem to the free-body diagram of the body as shown in figure we get

$$\frac{W}{\sin(90^\circ - \alpha + 30^\circ)} = \frac{P}{\sin(\pi - 30^\circ)} = \frac{R}{\sin(90^\circ + \alpha)}$$

Using the first two of the equation we obtain

$$\begin{aligned} \frac{W}{\cos(30^\circ - \alpha)} &= \frac{P}{\sin 30^\circ} \\ P &= \frac{W \sin 30^\circ}{\cos(30^\circ - \alpha)} \end{aligned}$$

From equation, P will be minimum, if the denominator is maximum, i.e.

$$\cos(30^\circ - \alpha) = 1$$

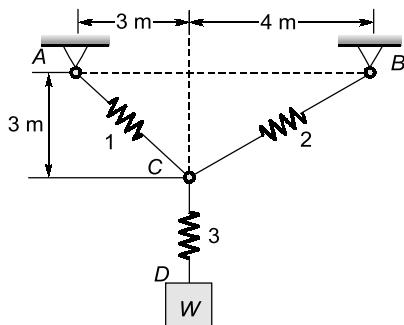
$$\Rightarrow 30^\circ - \alpha = 0$$

$$\Rightarrow \alpha = 30^\circ$$

and substituting this value into equation, we get the value of

$$P = W \sin 30^\circ = 150 \text{ N}, \text{ as before}$$

- Q.2** Determine the stretch in each spring for equilibrium of the weight $W = 40 \text{ N}$ block as shown in figure. The springs are in equilibrium position. The stiffness of each spring is given as: $k_1 = 40 \text{ N/m}$, $k_2 = 50 \text{ N/m}$, and $k_3 = 60 \text{ N/m}$



Solution:

Draw the free-body diagram of the body as shown in figure.

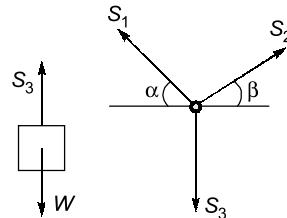
Only two forces are acting on the body, gravity force W and the reactive force caused by the spring S_3 . Since the body is in equilibrium, from the law of equilibrium of two forces,

$$S_3 = W$$

Now, draw the free-body diagram of the point C. At the joint, C three forces are acting all are reactive forces caused by the springs S_1 and S_2 . The angles that springs S_1 and S_2 make with the horizontal are calculated as below:

$$\tan \alpha = \frac{3}{3} = 1 \Rightarrow \alpha = 45^\circ$$

$$\tan \beta = \frac{3}{4} \Rightarrow \beta = 36.87^\circ$$



Since the joint C is in equilibrium, applying Lami's theorem, we obtain

$$\frac{S_1}{\sin\left(\frac{\pi}{2} + \beta\right)} = \frac{S_2}{\sin\left(\frac{\pi}{2} + \alpha\right)} = \frac{S_3}{\sin(\pi - \alpha - \beta)}$$

From equation we get

$$\Rightarrow S_1 = \frac{S_3 \cos \beta}{\sin(\alpha + \beta)} = \frac{W \cos \beta}{\sin(\alpha + \beta)}$$

$$S_2 = \frac{S_3 \cos \alpha}{\sin(\alpha + \beta)} = \frac{W \cos \alpha}{\sin(\alpha + \beta)}$$

$$EF = EC + CF = r_1 + r_2 = 100 + 50 = 150 \text{ mm}$$

and

$$EH = OI - OG - BI$$

$$OI = a = 200 \text{ mm}$$

and

$$OG = r_2 = 50 \text{ mm}$$

$$BI = EI \sin \frac{\alpha}{2} \left[\because EI = \frac{BE}{\cos \frac{\alpha}{2}} = \frac{r_1}{\cos 30^\circ} = \frac{100}{\cos 30^\circ} = 115.47 \text{ mm} \right]$$

$$\therefore$$

$$BI = 115.47 \sin 30^\circ = 57.74 \text{ mm and}$$

$$\therefore$$

$$EH = 200 - 50 - 57.74 = 92.26 \text{ mm}$$

$$\cos \beta = \frac{EH}{EF} = \frac{92.26}{150} = 0.615$$

\therefore

$$\beta = 52.05^\circ$$

$$R_c \cos \beta = R_d$$

$$R_c \sin \beta = Q$$

Substituting the values for β and Q in the above equations and solving for R_c and R_d , we obtain

$$R_c = \frac{Q}{\sin \beta} = \frac{800}{\sin 52.05^\circ} = 1014.52 \text{ N}$$

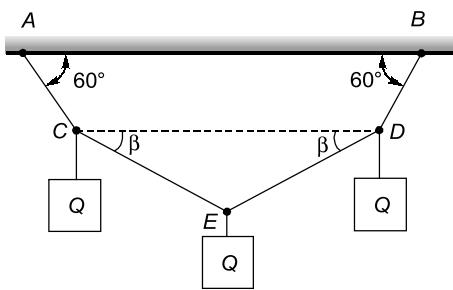
$$R_d = R_c \cos \beta = 1014.52 \times \cos 52.05^\circ = 623.9 \text{ N}$$

$$R_a = R_c \frac{\cos \beta}{\sin \alpha} = 1014.52 \times \frac{\cos 52.05^\circ}{\sin 60^\circ} = 720.42 \text{ N}$$

$$R_b = R_c \sin \beta + P - R_a \cos \alpha \\ = 1014.52 \times \sin 52.05^\circ + 2000 - 720.42 \cos 60^\circ = 2439.79 \text{ N}$$

- Q3** On the string $ACEDB$ are hung three equal weights Q symmetrically placed with respect to the vertical line through the mid-point E . Determine the value of the angles b if the other angles are as shown in the figure.

Solution:



At point E ,

By symmetry,

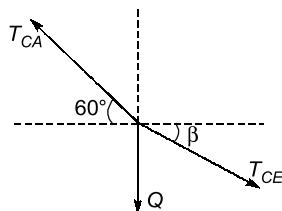
$$T_{CE} = T_{ED}$$

Lami's theorem

$$\frac{T_{CE}}{\sin(90 + \beta)} = \frac{T_{ED}}{\sin(90 + \beta)} = \frac{Q}{\sin(180 - 2\beta)}$$

$$T_{CE} = \frac{Q \cos \beta}{\sin 2\beta} = \frac{Q}{2 \sin \beta} \quad \dots(i)$$

At point C :

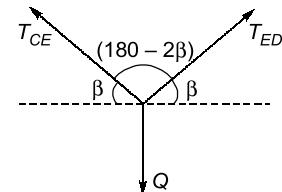
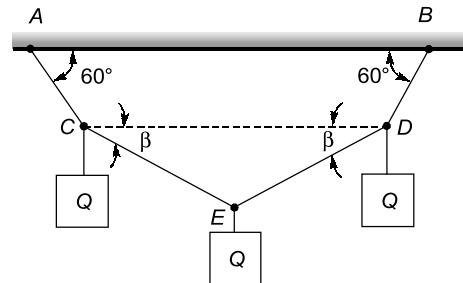


Lami's theorem

$$\frac{T_{CA}}{\sin(90 - \beta)} = \frac{T_{CE}}{\sin 150} = \frac{Q}{\sin(120 + \beta)}$$

Now,

$$T_{CE} = \frac{Q \times \sin 150}{\sin(120 + \beta)} \quad \dots(ii)$$



By equation (i) and (ii)

$$\frac{Q}{2\sin\beta} = \frac{Q \times 1/2}{\sin(120 + \beta)}$$

$$\sin(120 + \beta) = \sin \beta$$

$$\sin[90 + (30 + \beta)] = \sin \beta$$

$$\cos \beta \cdot \cos 30 - \sin \beta \cdot \sin 30 = \sin \beta$$

$$\cos \beta \times \frac{\sqrt{3}}{2} = \sin \beta + \frac{1}{2}(\sin \beta)$$

$$\frac{\cos \beta}{\sin \beta} = \frac{\frac{3}{2}}{\left(\frac{\sqrt{3}}{2}\right)}$$

$$\left(\frac{\cos \beta}{\sin \beta}\right) = \sqrt{3}$$

$$\tan \beta = \frac{1}{\sqrt{3}}$$

$$\beta = 30^\circ$$

Alternate:

$$\sin [180 - (120 + \beta)] = \sin \beta$$

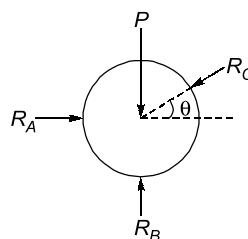
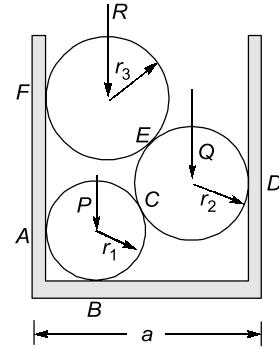
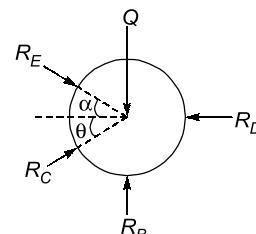
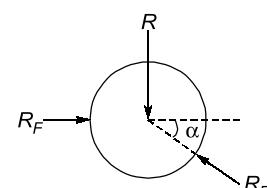
comparing on both sides

$$180 - 120 - \beta = \beta$$

$$60 = 2 \beta$$

$$\beta = 30^\circ$$

- Q4** The smooth cylinders rest in a horizontal channel having vertical walls, the distance between which is a . Find the pressures exerted on the walls and floor at the points of contact A, B, D and F . The following numerical data are given: $P = 200 \text{ N}$, $Q = 400 \text{ N}$, $R = 300 \text{ N}$, $r_1 = 120 \text{ mm}$, $r_2 = 180 \text{ mm}$, $r_3 = 150 \text{ mm}$ and $a = 540 \text{ mm}$.

Solution:

Cylinder 1

Cylinder 2

Cylinder 3

For cylinder 2:

$$\cos \alpha = \frac{540 - 180 - 150}{180 + 150}$$

$$\alpha = 50.47^\circ$$

$$\frac{R}{\sin(180 - 50.47)^\circ} = \frac{R_E}{\sin 90^\circ} = \frac{R_F}{\sin(90 + 50.47)^\circ}$$

$$R_E = \frac{R}{\sin(129.53^\circ)} = 388.96 \text{ N}$$

$$R_F = \frac{R \times \sin 140.47^\circ}{\sin(129.53^\circ)} = 247.565 \text{ N}$$

For cylinder 1:

$$\cos \theta = \frac{540 - 120 - 180}{120 + 180}$$

$$\theta = 36.87^\circ$$

Now $R_C \cos \theta = R_A \quad \dots(i)$

$$P + R_C \sin \theta = R_B \quad \dots(ii)$$

Now, $R_C = 1.25 R_A$

$$P + 0.6 R_C = R_B \Rightarrow P + 0.75 R_A = R_B$$

For cylinder 2: $Q + R_E \sin \alpha = R_C \sin \theta$

Now, $R_C = \frac{400 + 388.96 \times \sin 50.47^\circ}{\sin 36.87^\circ}$

$$R_C = 1166.67 \text{ N}$$

$$R_D = R_E \cos \alpha + R_C \cos \theta$$

$$R_D = 388.96 \cos 50.47^\circ + 1166.67 \cos 36.87^\circ$$

$$R_D = 1180.9 \text{ N}$$

For cylinder 1: $R_A = \frac{R_C}{1.25} = \frac{1166.67}{1.25} = 933.336 \text{ N}$

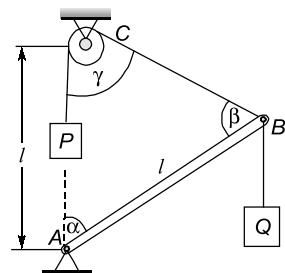
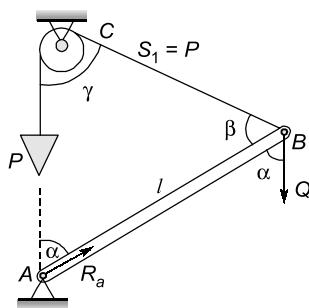
$$R_B = P + 0.75 R_A$$

$$= 200 + 0.75 \times 933.336$$

$$R_B = 900 \text{ N}$$

- Q5** A prismatic bar AB of negligible weight and length l is hinged at A and supported at B by a string that passes over a pulley C and carries a load P at its free end. Assuming that the distance h between the hinge A and the pulley C is equal to the length l of the bar, find the angle α at which the system will be in equilibrium.

Solution:



The triangle ABC is isosceles

$$\beta = \gamma = \frac{\pi - \alpha}{2} = 90^\circ - \left(\frac{\alpha}{2}\right)$$

Taking point A as the moment center (thus eliminating consideration of the unknown reaction at A), we obtain